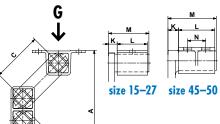
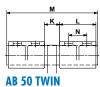
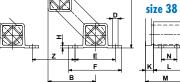


## **Oscillating Mountings**

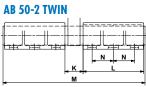
Type AB (standard blue) Type ABI (stainless steel)











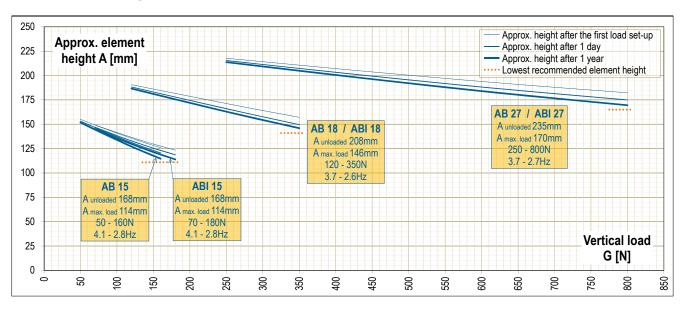
Art.	t. No.	Туре	Load ca Gmin. – (	Gmax.	A un- loaded	A* max. load	B un- loaded	B* max. load	С	D	E	F	Н	K	L	М	Z	Weight [kg]
07 0	051 056	AB 15	50 -	160	168	114	70	88	80	ø7	50	65	3	10	40	52		0.5
<mark>√</mark> 07 1	171 107	ABI 15	70 –	180	100	114	70	00	00	7x10	30	65	ა	10	40	32	_	0.9
07 0	051 057	AB 18	120 -	350	208	146	88	109	100	ø9	60	80	3.5	14	50	67	_	1.2
<mark>w</mark> ≻07 1	171 114	ABI 18	120 -	330	206	140	00	107	100	9x15	80	60	3.3	14	30	0/	_	1. <i>7</i>
07 0	051 058	AB 27	250 -	800	235	170	94	116	100	ø11	80	105	4.5	17	60	80	_	2.2
<mark>√</mark> 07 1	171 109	ABI 27	250 -	800	233	170	74	110	100	11x20	80	103	4.5	17	00	60	_	3.3
07 0	051 059	AB 38	600 -	1′600	305	225	120	147	125	ø13	100	125	6	21	80	104	40	5.1
<mark>w</mark> ≻07 1	171 110	ABI 38	800 -	1 000	303	223	120	14/	123	13x20	100	123	0	21	80	104	40	7.6
07 0	051 054	AB 45	1′200 –	3′000	353	257	141	172	140	13×26	115	145	8	28	100	132	58	11.5
<mark>√</mark> 07 1	1 <i>7</i> 1 111	ABI 45	1 200 -	3 000	333	237	137	168	140	13 X Z O	113	143	O	20	100	132	50	13.5
07 0	051 061	AB 50	2′500 –	6′000	380	277	150	184	150	17×27	130	170	12	35	120	160	60	19.1
<mark>√</mark> 07 1	171 112	ABI 50	2 300 -	8 000	360	2//	150	104	130	1/ X Z/	130	170	12	33	120	100	00	21.9
07 0	051 055	AB 50-2	4′200 –	10′000	380	277	150	184	150	17×27	130	170	12	40	200	245	70	32.2
<mark>w</mark> ≻07 1	171 113	ABI 50-2	4 200 -	10 000	360	2//	130	104	130	1/ X Z/	130	170	12	40	200	243	/0	35.4
07 0	051 008	AB 50 TWIN	5′000 -	12′000	380	277	150	184	150	17×27	130	170	12	50	120	300	60	35.0
07 0	051 009	AB 50-2 TWIN	8′400 –	20′000	380	277	150	184	150	17×27	130	170	12	60	200	470	70	54.0

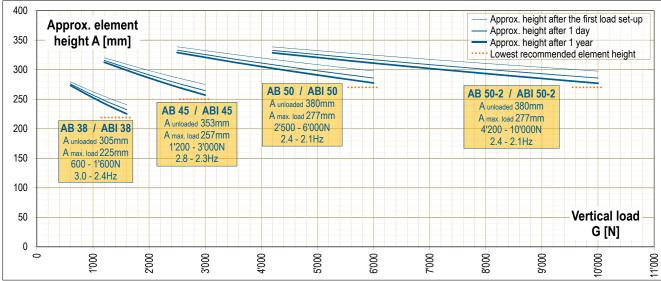
			Dynamic spring value			Capacity limits by different rpm						ofile		inted	casting	
						720 min <sup>-1</sup>		960 min <sup>-1</sup>		1440 min <sup>-1</sup>		pro	ed c	ast i	pd e	ω Jei
Art. No.	Туре	Natural frequency GminGmax. [Hz]	Z	cd vertical [N/mm]	cd horizontal [N/mm]	sw max. [mm]	K max. [–]	sw max. [mm]	K max. [–]	sw max. [mm]	K max. [-]	Light metal profile	Steel welded construction	Nodular cast iron	ROSTA blue painted	Stainless steel
07 051 056	AB 15				. , .							x	x		x	
<b>~</b> 07 171 107	ABI 15	4.0 – 2.8	65	10	6	14	4.1	12	6.2	8	9.3					x
07 051 057	AB 18											х	х		х	
<b>~</b> ≥07 171 114	ABI 18	3.7 – 2.6	80	20	14	17	4.9	15	7.7	8	9.3					х
07 051 058	AB 27	27 27	00	40	0.5	17	4.0	1.4	70	_	9.3	х	х		х	
<b>√</b> 07 171 109	ABI 27	3.7 – 2.7	80	40	25	17	4.9	14	7.2	8	9.3					х
07 051 059	AB 38	3.0 – 2.4	100	60	30	20	5.8	17	8.8	8	9.3	х	х		х	
<b>~</b> −07 171 110	ABI 38	3.0 - 2.4	100	00		20	J.0	17	0.0	0	7.5					х
07 051 054	AB 45	2.8 – 2.3	115	100	50	21	6.1	18	9.3	8	9.3	х	Х	х	х	
07 171 111	ABI 45	2.0 2.0	110	100				,0	7.0	J	7.5					х
07 051 061	AB 50	2.4 – 2.1	140	190	85	22	6.4	18	9.3	8	9.3			х	х	
07 171 112	ABI 50			., 0			<b>0.</b> .		7.0	ŭ	7.0					х
07 051 055	AB 50-2	2.4 – 2.1	140	320	140	22	6.4	18	9.3	8	9.3			х	х	
07 171 113	ABI 50-2															Х
07 051 008	AB 50 TWIN	2.4 – 2.1	140	380	170	22	6.4	18	9.3	8	9.3		Х	Х	х	
07 051 009	AB 50-2 TWIN	2.4 – 2.1	140	380	170	22	6.4	18	9.3	8	9.3		Х	Х	Х	
ROSTA 🚮				Values in nominal load range at 960 min <sup>-1</sup> and sw of 8 mm		Acceleration > 9.3 g is not recommended				Material structure						

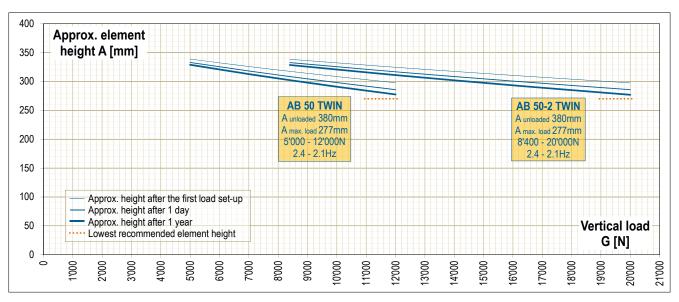


<sup>\*</sup> compression load Gmax. and cold flow compensation (after approx. 1 year).

## Element heights and cold flow behaviour AB and ABI

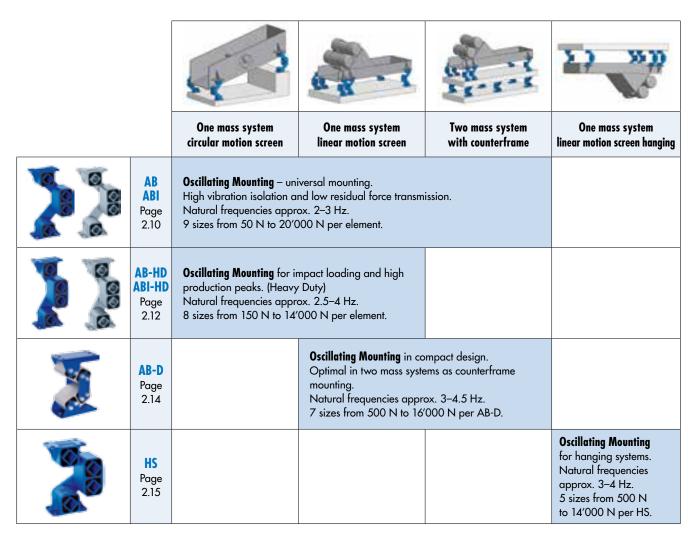




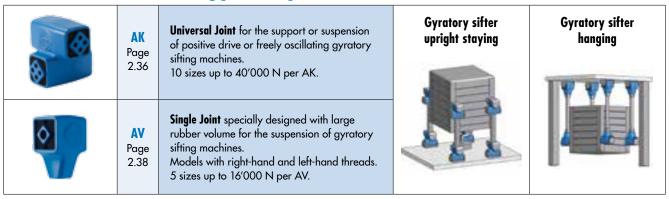




# Selection table for free oscillating systems (with unbalanced excitation)



## Selection table for gyratory sifters





# Technology of free oscillating systems with unbalanced excitation

### Introduction

Free oscillating systems are either activated in using exciters, unbalanced motors or unbalanced shafts.

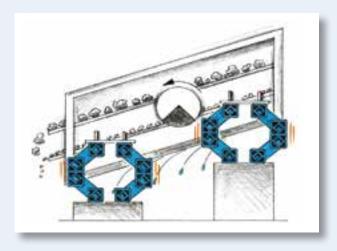
The oscillation amplitude, type of vibration and the direction of vibration of the screen are determined by the dimensioning and arrangement of these actuators. The excitation force, the angle of inclination of the excitation, the inclination of the screen-box and the position of the center of gravity determine the resulting oscillation amplitude of the device. The oscillation amplitude, and thereby the conveying speed of the machine, can be optimized by augmenting these.

ROSTA spring suspensions support the desired oscillation movement of the screen machine. Through their shape and function, they help to achieve a purely linear conveyor motion without unwanted lateral tumbling.

These ideal spring suspensions harmonically support the running of the vibrating screen. Because of their high spring deflection capacity, they offer a good detuning of the excitation frequency with a very low natural frequency, which guarantees a high isolation effect with regard to the machine substructure. The ROSTA mounts effectively dissipate the large residual force peaks at start-up and shut-down, when passing through the natural frequency of the suspension



## **Circular motion screens**

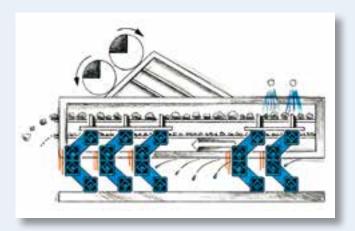


Circular motion screens or circular vibrators are normally excited by unbalanced weights that create a circular rotating oscillation of the screening frame. Relatively low accelerations of the screened material are achieved with this form of excitement. Circular vibrators thereby normally work with a screening frame inclination of 15° to 30°, so that an adequate material throughput is ensured.

It is recommended to mount circular vibratory screens of this kind on ROSTA type AB or AB-HD oscillating mountings. Experience has shown that the positioning of the AB suspensions under circular vibrators should be a mirror-inverted of each other, which, with the above-mentioned frame inclination, will counteract the tendency of the shifting of the center of gravity. If the suspension of the screening frame requires two supporting suspensions per brace support for reasons of capacity, these should also be preferably arranged in mirror-inverted manner for the above-mentioned reason.



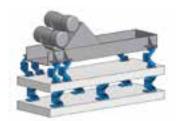
### **Linear motion screens**



Linear motion screens or linear vibrators are normally excited by two unbalanced motors or by means of linear exciters, as well as through double unbalanced shafts (Eliptex), which generate a linear or slightly elliptical oscillation of the screening frame. Depending on the inclination positioning of the exciter, the angle of throw of the screened product can be adapted to the desired form of processing. A very high acceleration of the screened product, i.e. a higher material throughput, is achieved with linear vibrating screens. The screening frame of the linear vibrator is normally in the horizontal position.

Linear vibrating screens are preferably mounted on ROSTA oscillating mountings type AB or AB-HD. Depending on the positioning of the exciter on the screening frame, the feed-end: discharge-end load distribution can be different. The feed-end side is normally lighter, as the exciters are positioned close to the discharge-end and thereby pull the material through the screening frame; in many cases, the feed-end: discharge-end distribution is thereby 40% to 60%. In the interest of an even suspension, it is thereby recommended to mount the screening frame on six or more ROSTA oscillating mountings. All oscillating mountings should stand in the same direction, with the "knee" pointing in the discharge-end direction.

# Linear motion screens with counterframe

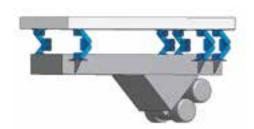


If, due to the demands of the process, large screens are mounted at a very high position in a building or in a purely steel construction, the transmission of the residual forces of a singlemass machine can set the

entire structure into unwanted vibrations. Or if a new and more powerful machine is mounted in an existing building, the residual force transmission could be too high for the older building. The residual force transmission is drastically reduced through the mounting of a counterframe under the screen, with only a negligible loss of oscillation amplitude (compensation movement of the counterframe reduces the oscillation amplitude).

ROSTA also has the ideal supports for the suspension of counterframes, the very compact mountings type AB-D.

# Discharge chutes hanging under silos and bunkers



Discharge chutes under silos are normally supported by means of complicated yoke constructions and are suspended on pressure springs. With its HS suspensions (HS = hanging screen), ROSTA offers the possibility of the direct, costeffective suspension of the discharge unit on silos and bunkers. The geometry of the HS suspensions has been designed to accommodate tensile loads.



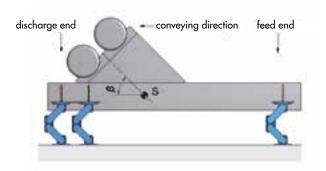
## **Technology**

### **Design layout and evaluation**

• Natural frequency suspensions fe

Degree of isolation

Mass of the empty channel and drive Products on the channel 200 kg of which approx. 50% coupling* 100 kg  Total vibrating mass* m 780 kg  Mass distribution: feed end discharge end 8 discharge end 67 % Acceleration due to gravity g 9.81 m/s²  Load per corner feed end Ffeed end 1263 N  Load per corner discharge end Fdischarge end 2563 N  • Element choice in example 6 × AB 38  Working torque of both drives AM 600 kgcm  Oscillating stroke empty channel swo 8.8 mm  Oscillating stroke in operation sw 7.7 mm  Motor revolutions Ps 20319 N  Oscillating machine factor K 4.0  Machine acceleration a = K · q 4.0	Products on the channel 200 kg of which approx. 50% coupling* 100 kg  Total vibrating mass* m 780 kg  Mass distribution: feed end discharge end discharge end discharge end 67 %  Acceleration due to gravity g 9.81 m/s²  Load per corner feed end Ffeed end 1263 N  Load per corner discharge end Fdischarge end 2563 N  • Element choice in example 6x AB 38  Working torque of both drives AM 600 kgcm  Oscillating stroke empty channel swo 8.8 mm  Oscillating stroke in operation sw 7.7 mm  Motor revolutions ns 960 rpm  Centrifugal force of both drives Fz 30′319 N  Oscillating machine factor K 4.0	Subject		Symbol	• Example	Unit
discharge end %discharge end 67 %  Acceleration due to gravity g 9.81 m/s²  Load per corner feed end Feed end 1263 N  Load per corner discharge end 5 Fdischarge end 2563 N  • Element choice in example 6 × AB 38  Working torque of both drives AM 600 kgcm  Oscillating stroke empty channel swo 8.8 mm  Oscillating stroke in operation sw 7.7 mm  Motor revolutions ns 960 rpm  Centrifugal force of both drives Fz 30′319 N  Oscillating machine factor K 4.0	discharge end %discharge end 67 %  Acceleration due to gravity g 9.81 m/s²  Load per corner feed end Ffeed end 1263 N  Load per corner discharge end Fdischarge end 2563 N  • Element choice in example 6x AB 38  Working torque of both drives AM 600 kgcm  Oscillating stroke empty channel sw0 8.8 mm  Oscillating stroke in operation sw 7.7 mm  Motor revolutions ns 960 rpm  Centrifugal force of both drives Fz 30′319 N  Oscillating machine factor K 4.0	Products on the ch of which approx.	annel 50% coupling*	Ü	200 100	kg kg
Oscillating stroke empty channel sw <sub>0</sub> 8.8 mm  Oscillating stroke in operation sw 7.7 mm  Motor revolutions n <sub>s</sub> 960 rpm  Centrifugal force of both drives Fz 30'319 N  Oscillating machine factor K 4.0	Oscillating stroke empty channel sw <sub>0</sub> 8.8 mm  Oscillating stroke in operation sw 7.7 mm  Motor revolutions n <sub>s</sub> 960 rpm  Centrifugal force of both drives Fz 30'319 N  Oscillating machine factor K 4.0	Acceleration due t Load per corner fe Load per corner d	discharge end o gravity eed end ischarge end	% discharge end g F feed end	67 9.81 1263 2563	% m/s <sup>2</sup> N
		Oscillating stroke	empty channel	sw <sub>0</sub>	8.8	mm



#### **Calculation formulas**

#### Loading per corner

$$F_{\text{feed-end}} = \frac{\text{m} \cdot \text{g} \cdot \text{\% feed-end}}{2 \cdot 100} \quad F_{\text{discharge-end}} = \frac{\text{m} \cdot \text{g} \cdot \text{\% discharge-end}}{2 \cdot 100} \; \left[ \; N \; \right]$$

#### Oscillating stroke (Amplitude peak to peak)

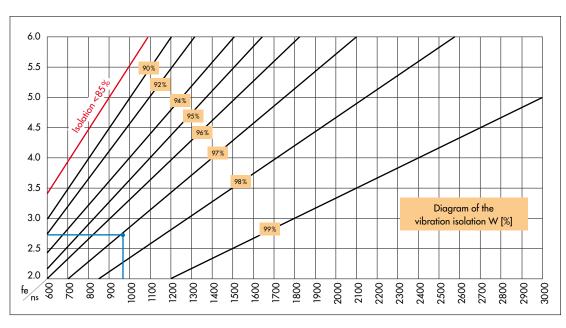
$$sw_0 = \frac{AM}{m_0} \cdot 10$$
  $sw = \frac{AM}{m} \cdot 10$  [ mm ]

#### **Centrifugal force**

$$F_z = \frac{\left(\frac{2\pi}{60} \cdot n_s\right)^2 \cdot AM \cdot 10}{2 \cdot 1000} = \frac{n_s^2 \cdot AM}{18'240} [N]$$

#### Oscillating machine factor

$$K = \frac{\left(\frac{2\pi}{60} \cdot n_s\right)^2 \cdot sw}{2 \cdot g \cdot 1000} = \frac{n_s^2 \cdot sw}{1'789'000} [-]$$



2.7 Hz

#### **Vibration** isolation

W = 100 - 
$$\frac{100}{\left(\frac{n_s}{60 \cdot f e}\right)^2 - 1} [\%]$$

#### Example:

The proportion of the relationship between exciter frequency 16 Hz (960 rpm) and mount frequency 2.7 Hz is offering a degree of isolation of 97%.

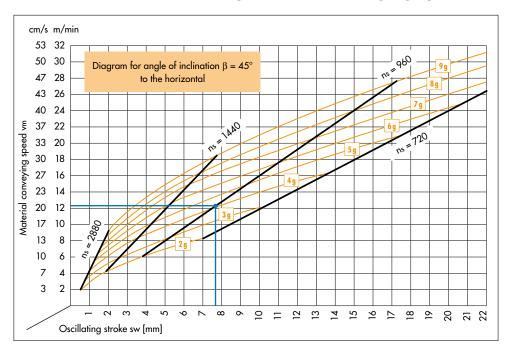
- \* The following has to be observed for the determination of the coupling effect and material flow:
- High coupling or sticking of humid bulk material
- Channel running full
- Fully stacked screen deck with humid material
- Weight distribution with and without conveyed material
- Centrifugal force does not run through the center of gravity (channel full or empty)
- Sudden impact loading occurs
- Subsequent additions to the screen structure (e.g. additional screening deck)





## **Technology**

### Determination of the average material conveying speed vm



#### Main influencing factors:

- Conveying ability of the material
- Height of the bulk goods
- Screen box inclination
- Position of unbalanced motors
- Position of the center of gravity

The material speed on circular motion screens does vary, due to differing screen-box inclination angles.

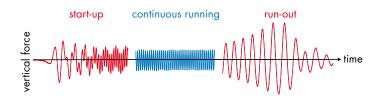
#### • Example:

The horizontal line out of the intercept point of stroke (7.7 mm) and motor revolutions (960 rpm) is indicating an average theoretical speed of 12.3 m/min or 20.5 cm/sec.

# Resonance amplification and continuous running

At the screen start-up and run-out the suspension elements are passing through the resonance frequency. By the resulting amplitude superelevation the four rubber suspensions in the AB mountings do generate a high level of damping which is absorbing the remaining energy after only a few strokes. The screen box stops its motion within seconds.

Laboratory measurements of a typical development of the residual forces on a ROSTA screen suspension:



### Alignment of the elements

If the suspensions for linear motion screens are arranged as shown on page 2.7, a harmonic, noiseless oscillation of the screen will result. The rocker arm fixed to the screen carries out the greater part of the oscillations. The rocker arm fixed to the substructure remains virtually stationary and ensures a low natural frequency, and thereby also a good vibration isolation. The mounting axis has to be arranged to be at right angles  $(90^{\circ})$  to the conveying axis, with maximum tolerance of  $\pm 1^{\circ}$ .

